A Mixed-Integer Programming Approach to Design of Periodic Structures with Negative Thermal Expansion

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structures with negative thermal expansion

• ...contract when heated.



- high (positive) TE
- low (positive) TE

mixed-integer programming

• m-i linear prog.:

$$\begin{array}{ll} \min \quad \boldsymbol{c}^{\mathrm{T}}\boldsymbol{x} + \boldsymbol{r}^{\mathrm{T}}\boldsymbol{y} \\ \mathrm{s.\,t.} \quad \boldsymbol{a}_{i}^{\mathrm{T}}\boldsymbol{x} + \boldsymbol{g}_{i}^{\mathrm{T}}\boldsymbol{y} \geq b_{i} \quad (i = 1, \ldots, m), \\ \quad \boldsymbol{x} \in \{0, 1\}^{n}, \quad \boldsymbol{y} \in \mathbb{R}^{l} \end{array}$$

• "mixed":

- x_j : integer (discrete) variable
- y_l : real (continuous) variable
- replace $x \in \{0,1\}^n$ with $0 \le x \le 1$ \rightarrow linear prog. relaxation
- can be solved with, e.g., a branch-and-bound method

negative thermal expansion

- materials with NTE
 - zirconium tungstate family

[Martinek & Hummel '68]

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- (possible) applications
 - to reduce thermal stress in a structure
 - thermal fastener

[Sigmond & Torquato '96]

- inserted into a hole at high temperature
- fitted tightly into the hole when it cools down
- sensitive temperature sensors
 [Sleight '98]
 - combination of two thin films with large PTE and large NTE

- existing methods:
 - continuum & homogenization method

[Sigmund & Torquato '97], [Chen, Silva, & Kikuchi '01]

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- Manual post-processing before fabrication: de facto hinges.
 - fabrication of opt. sol.
 - direct metal deposition: powdered metals are melted by laser [Chen, Silva, & Kikuchi '01]
 - oxide co-extrusion: powder based thermoplastic processing [Qi & Halloran, 04]

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[Sigmund & Torquato '97], [Chen, Silva, & Kikuchi '01]

- our method:
 - periodic frame structure
 - stress constraints & pre-determined beam sections

 \rightarrow no hinge, no thin member



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- \rightarrow manufacturability (no post-processing)
- ullet ightarrow global optim.
 - an idea MILP for truss
 [Rasmussen & Stolpe '08] [K. & Guo '10]

problem setting

- assume periodicity & symmetry
- unit cell: planar frame structure
- design variables: sections of members



problem setting

- design of base cell
 - prepare candidate members, e.g.,



problem setting

- design of base cell
 - prepare candidate members, e.g.,



- two materials (w/ different positive TECs)
 - <u>Determine</u>

member
$$e = \begin{cases} material 1 \\ material 2 \\ void \end{cases}$$

so as to minimize u_{temp} due to temperature elevation.

optimization problem

max	$u_{ ext{temp}}$
s.t.	equil. eq. at elevated temperature
	stress constraints
	avoiding member intersection, etc.

- integer variables
 - for member *i*,

$$(x_e, y_e) = (1, 0) \Leftrightarrow e \in M_1 \text{ (material 1)}$$
$$(x_e, y_e) = (0, 1) \Leftrightarrow e \in M_2 \text{ (material 2)}$$
$$(x_e, y_e) = (0, 0) \Leftrightarrow e \in N \text{ (void)}$$

ullet ightarrow can be reduced to MIP.

reduction to MIP (1)

• stiffness matrix:

$$K = \sum_{e=1}^{m} \sum_{j=1}^{3} k_{et} \boldsymbol{b}_{et} \boldsymbol{b}_{et}^{\mathrm{T}}$$
 (\boldsymbol{b}_{et} : const. vec.

• member stiffnesses:

$$k_{et} = \bar{k}_{et1} x_e + \bar{k}_{et2} y_e \qquad (\bar{k}_{etp} : \text{ const.})$$

• member TECs:

 $\alpha_e = \bar{\alpha}_{e1} x_e + \bar{\alpha}_{e2} y_e$

$$(\bar{\alpha}_{ep}: \text{ const.})$$

• integer variables for member e:

$$x_e = \begin{cases} 1 & \text{if } M_1 \text{ is used} \\ 0 & \text{otherwise} \end{cases} \quad y_e = \begin{cases} 1 & \text{if } M_2 \text{ is used} \\ 0 & \text{otherwise} \end{cases}$$

reduction to MIP (2)

• thermoelastic equil. eq. $K \boldsymbol{u} = \boldsymbol{f}(\Delta t) \Leftrightarrow$

$$\sum_{e=1}^{m} \sum_{j=1}^{3} s_{et} \boldsymbol{b}_{et} = \boldsymbol{0} \qquad \text{(force-balance)}$$

$$s_{et} = \begin{cases} \bar{k}_{et1} (\boldsymbol{b}_{et}^{\mathrm{T}} \boldsymbol{u} - \bar{\alpha}_{e1} \Delta T) & \text{if } x_e = 1 & (\diamondsuit 1) \\ \bar{k}_{et2} (\boldsymbol{b}_{et}^{\mathrm{T}} \boldsymbol{u} - \bar{\alpha}_{e2} \Delta T) & \text{if } y_e = 1 & (\diamondsuit 2) \\ 0 & \text{otherwise} \quad (\clubsuit) \end{cases}$$

• stress constraints:

$$|q_e(\boldsymbol{u})|/q_e^{\mathrm{y}} + |m_e(\boldsymbol{u})|/m_e^{\mathrm{y}} \le 1$$

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(
$$\clubsuit$$
) & (\diamondsuit) \Leftrightarrow $|s_{e1}|/s_{e1}^{\max} + \dots + |s_{e3}|/s_{e3}^{\max} \le x_e + y_e$

$$\begin{array}{ll} (\diamondsuit 1) & \Leftrightarrow & |s_{et} - \bar{k}_{et1} (\boldsymbol{b}_{et}^{\mathrm{T}} \boldsymbol{u} - \bar{\alpha}_{e1} \Delta T)| \leq M(1 - x_{e}) \\ & (M \gg 0 : \text{const.}) \end{array}$$

goal: MIP formulation

max u_{temp} m 3 s.t. $\sum \sum s_{et} \boldsymbol{b}_{et} = \boldsymbol{f},$ e=1 i=1 $|s_{et} - \bar{k}_{et1}(\boldsymbol{b}_{et}^{\mathrm{T}}\boldsymbol{u} - \bar{\alpha}_{e1}\Delta T)| \leq M(1 - x_e), \quad \forall t, \forall e,$ $|s_{et} - \bar{k}_{et2}(\boldsymbol{b}_{et}^{\mathrm{T}}\boldsymbol{u} - \bar{\alpha}_{e2}\Delta T)| \leq M(1 - y_e), \quad \forall t, \ \forall e,$ $\sum_{t=1}^{3} \frac{|s_{et}|}{s_{et}^{\max}} \le x_e + y_e,$ $\forall i$, $x_e, y_e \in \{0, 1\},\$ $\forall i.$

- 66 candidate members
- Timoshenko beam element
- solver: CPLEX ver. 12.2
- $u_{\text{temp}} \to \min$
- beam cross-section
 - (width) \times (thickness) = $1 \times 1 \text{ mm}$





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towards large-scale examples

- use heuristics implemented in the MILP solver
- lose guarantee of global optimality
- CPLEX option
 - emphasis mip = 4

(emphasize to find hidden feasible solutions)

• timelimit = $1,000\,\mathrm{s}$

(limit run time, report the best feasible solution)

- \rightarrow Global opt. sols. were found $\,$ for all the examples above.
- After that, e.g., 66 h were spent to prove global optimality.
- try larger examples with this heuristic...

more examples



more examples



- Small TECs are achieved.
- Two materials are placed alternately. \rightarrow Explore "simpler" solutions.



separation of material distributions: motivation



• compressions act on interfaces between M_1 and M_2

 $\bullet \rightarrow$ Bonding will be strengthened when temperature elevates.

separation of material distributions: formulation



- 0-1 variables
 - x_e : indicates existence of member e
 - z_i : classifies node i
 - if node i : interior
 & node j : exterior

$$\Rightarrow z_i \leq z_j$$

• notation:
$$e = (i, j)$$

• material selection rule:

$$x_e = 1, \ z_i + z_j \le 1 \quad \Leftrightarrow \quad e \in \mathcal{M}_1,$$

$$x_e = 1, \ z_i + z_j = 2 \quad \Leftrightarrow \quad e \in \mathcal{M}_2,$$

$$x_e = 0, \ z_i + z_j \le 2 \quad \Leftrightarrow \quad e \in \mathcal{N}.$$





 $-2.12746 \times 10^{-5} \,\mathrm{K^{-1}}$ (global, 9,500 s)







 $0.08285 \times 10^{-5} \,\mathrm{K^{-1}}$ (12 h)





 $-0.04296 imes 10^{-5} \, {
m K}^{-1}$ (12 h)

- fabrication
 - lattice consisting of M_2 (exterior)
 - pieces consisting of M_1 (interior)
 - heating M_2 and inserting M_1

 \rightarrow Bonding will be strengthened automatically.





summary

- design of counterintuitive structures
 - \rightarrow Optimization might be a helpful tool.
- structures with negative thermal expansion
 - topology optimization of frame structures
 - min. the displacement at elevated temperature
 - mixed-integer programming
 - selection of material of member e
- \leftarrow integer variables
- two materials (w/ positive TECs) and void
- stress constraints
- no hinge, no thin member, no post-processing